Application No. 10/647,133 Amendment dated April 12, 2006

Reply to Office Action of December 13, 2005

Amendments to the Claims:

This listing of claims will replace all prior versions, and listings of claims in

the application:

1. (currently amended) A seal assembly for a rotary shaft, having a surface speed

higher than 200 ft/min, the seal assembly comprising: a housing having a generally

cylindrical internal surface receiving the shaft, said internal surface including first and

second internal annular grooves spaced longitudinally of and encircling said internal

surface; an O-ring disposed in each groove and in contact with the external surface of

the shaft and against which the shaft rotates in use; inlet and outlet ports for a cooling

fluid extending through the housing into said generally cylindrical internal surface of the

housing at locations spaced angularly about the housing and between said O-rings,

whereby an annular chamber is defined between the shaft and the housing intermediate

the two O-rings, through which a cooling fluid is circulated in use via said ports; and

means for creating a positive cooling fluid pressure within said chamber; the O-rings

being deformable under said pressure to seal against the rotary shaft.

2. (previously presented) A seal assembly as claimed in claim 1, wherein said generally

cylindrical internal surface of the housing is enlarged radially outwardly in the area of

said annular chamber as compared with the diameter of the surface outwardly of said

grooves.

3. (previously presented) A seal assembly as claimed in claim 1, wherein said external

surface of the shaft is defined by a replaceable sleeve which is received within said

generally cylindrical internal surface of the seal housing in contact with said O-rings.

2

Application No. 10/647,133

Amendment dated April 12, 2006

Reply to Office Action of December 13, 2005

4. (previously presented) A seal assembly as claimed in claim 1 in combination with a conventional centrifugal pump, wherein said housing has a generally cylindrical external

surface and is dimensioned to be received in a seal housing of said pump.

5. (previously presented) A seal assembly as claimed in claim 4, wherein said housing

includes an outwardly projecting annular flange at an outer end of the housing, the

flange having an inner surface provided with a gasket for sealing against a

corresponding face of said seal housing.

6. (previously presented) A seal assembly as claimed in claim 5, wherein said

cylindrical housing has an annular end face at an end of the housing remote from the

flange, said face being provided with a gasket for sealing internally within the seal

housing.

7. (previously presented) A seal assembly as claimed in claim 5, further comprising an

annular lip seal extending inwardly of the housing at the location of said flange.

8. (previously presented) A seal assembly as claimed in claim 1, wherein said fluid inlet

and outlet ports are disposed generally in diametrally opposed locations around said

housing for communication respectively with a supply of said fluid and a throttle valve

which is operable to control fluid flow from said outlet port, said throttle valve comprising

said means for producing a positive cooling fluid pressure within said chamber.

9. (cancelled)

10. (currently amended) A method of sealing a rotary shaft against fluid leakage along

the shaft, wherein the shaft rotates at a surface speed higher than 200 ft/min and

wherein the method comprises comprising the steps of:

providing a seal assembly comprising: a housing having a generally

cylindrical internal surface which receives the shaft and which includes first and second

3

Application No. 10/647,133
Amendment dated April 12, 2006
Reply to Office Action of December 13, 2005

internal annular grooves spaced longitudinally of and encircling the housing; an O-ring disposed in each groove and in contact with the external surface of the shaft and against which the shaft rotates in use; and an annular chamber defined between the shaft and the housing intermediate the two O-rings; and,

circulating a cooling fluid through said annular chamber under pressure to cause the O-rings to deform and seal against the rotary shaft.

- 11. (original) A method as claimed in claim 10, wherein said step of circulating a cooling fluid through said annular chamber under pressure comprises introducing fluid into the chamber from an inlet at a supply pressure and conducting fluid from the chamber through an outlet while throttling the flow of fluid to provide a desired fluid pressure within the chamber.
- 12. (original) A method as claimed in claim 11, comprising the further step of monitoring leakage along the shaft and, in the event that leakage is detected, reducing the flow of fluid from said outlet so as to increase pressure within the chamber.
- 13. (original) A method as claimed in claim 10, wherein the fluid circulated through the chamber is water and the pressure in the chamber is in the range 20 30 psi.
- 14. (original) A method as claimed in claim 13, wherein the water is circulated through the chamber at a flow rate of 5 to 10 imperial gallons per minute.
- 15. (cancelled)
- 16. (previously presented) A method of sealing a rotary drive shaft of a centrifugal pump against fluid leakage along the shaft, the method comprising the steps of:

providing a seal assembly comprising: a housing having a generally cylindrical internal surface which receives the shaft and which includes first and second internal annular grooves spaced longitudinally of and encircling the housing; an O-ring

Application No. 10/647,133 Amendment dated April 12, 2006 Reply to Office Action of December 13, 2005

disposed in each groove and in contact with the external surface of the shaft and against which the shaft rotates in use; and an annular chamber defined between the shaft and the housing intermediate the two O-rings;

rotating said shaft at a linear surface speed in a range from 200 ft/min to 4,713 ft/min; and,

circulating a cooling fluid through said annular chamber under pressure to cause the O-rings to deform and seal against the rotary shaft.